

# Numerical Solution to 1-D Steady Conduction / ME516 Sp 2013 3.1

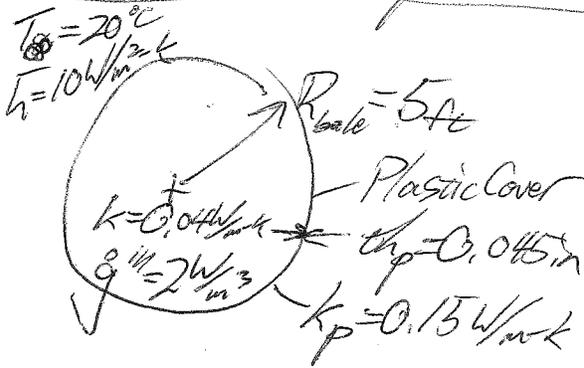
→ Last time we introduced analytical solutions to Steady-State 1-D conduction problems.

→ Analytical solutions are very nice  
 \* they are exact. \* they are computationally fast

- Limits to analytical
- Integrated average k
- Numerical solution
- Define nodal network
- Energy balance(s)
- rate equations
- numerical convergence
- post analysis

→ Problem is, Analytical solutions are not flexible.

Back to the Haybale Example: when we solved this we assumed



that properties in the hay bale are uniform, <sup>both</sup> throughout the bale & @ any temp. The thermal conductivity & volumetric generation are likely not constant, & probably functions of both temperature & position due to moisture content & density

our ODE from last time:

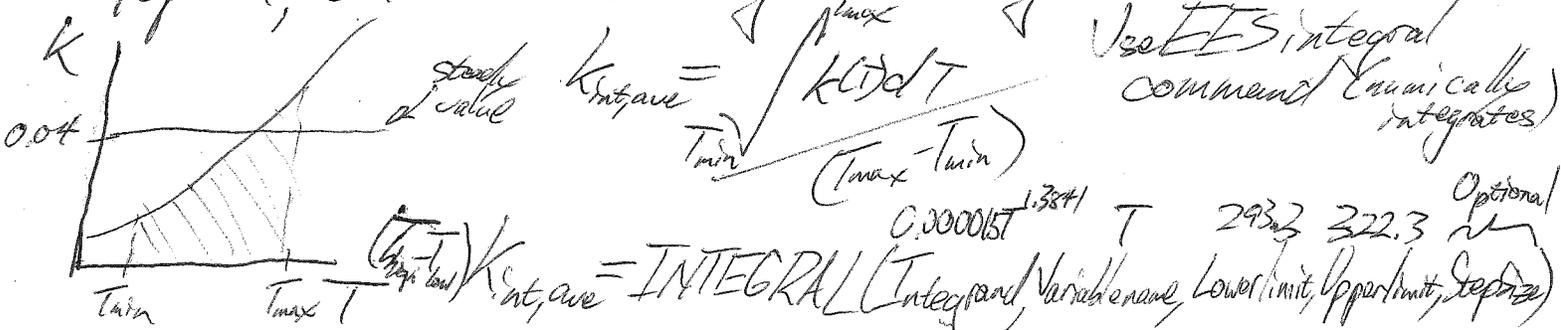
$$r^2 q''' = \frac{d}{dr} \left( k r \frac{dT}{dr} \right)$$

~~0.075~~ 0.075

what if  ~~$q''' = 0.01T - 1$~~   $q''' = 0.01T - 1$  &  ~~$k = 0.000015T^{1.3841}$~~   $k = 0.000015T^{1.3841}$  ?

With just  $q'''$  we can still get analytical solution, but it involves Bessel functions, with  $k$  function it's just way too complicated,

→ option, calculate the integrated average values

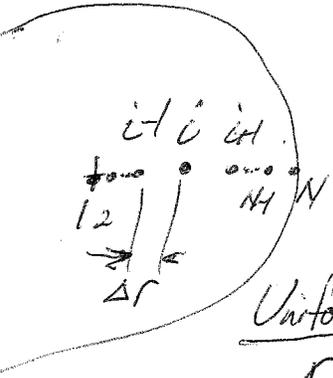


Plug into EES  $k_{ave} = \frac{0.04171}{\ln \frac{t_{high}}{t_{low}}}$   $k_{high} = 0.0444$   $k_{low} = 0.039$   
~~our analytical solution~~

This might get us closer, but the spatial effects on  $T$  are not represented by the varying  $k$ , & we'll have to iterate the solution for  $T_{bounds}$ . Really we need a different technique.

Numerical Solutions: Provide predictions of the temperature @ discrete locations (nodes) within the computational domain.

- Numerical Solutions are approximate → Computationally slower
- Flexible - easy to include temperature dependent properties



Step 1: Define many ( $N$ ) nodes (locations where you will predict temperature) & the associated control volumes.

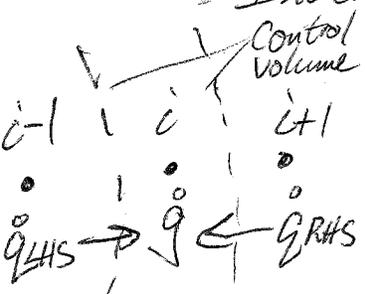
→ Small, but finite ( $\Delta x$  is finite vs.  $\Delta x$  approaches zero)  
Uniformly distributed nodes

$$r_i = \frac{(i-1)}{(N-1)} R_{Total} \quad \text{for } i=1 \rightarrow N \quad \Delta r = \frac{R_{Total}}{(N-1)}$$

→ You can also write functions to concentrate nodes in regions of interest (e.g., regions with large temperature gradients)

Step 2: Carry out an energy balance on each control volume

→ Internal nodes will look similar → boundaries must be treated separately

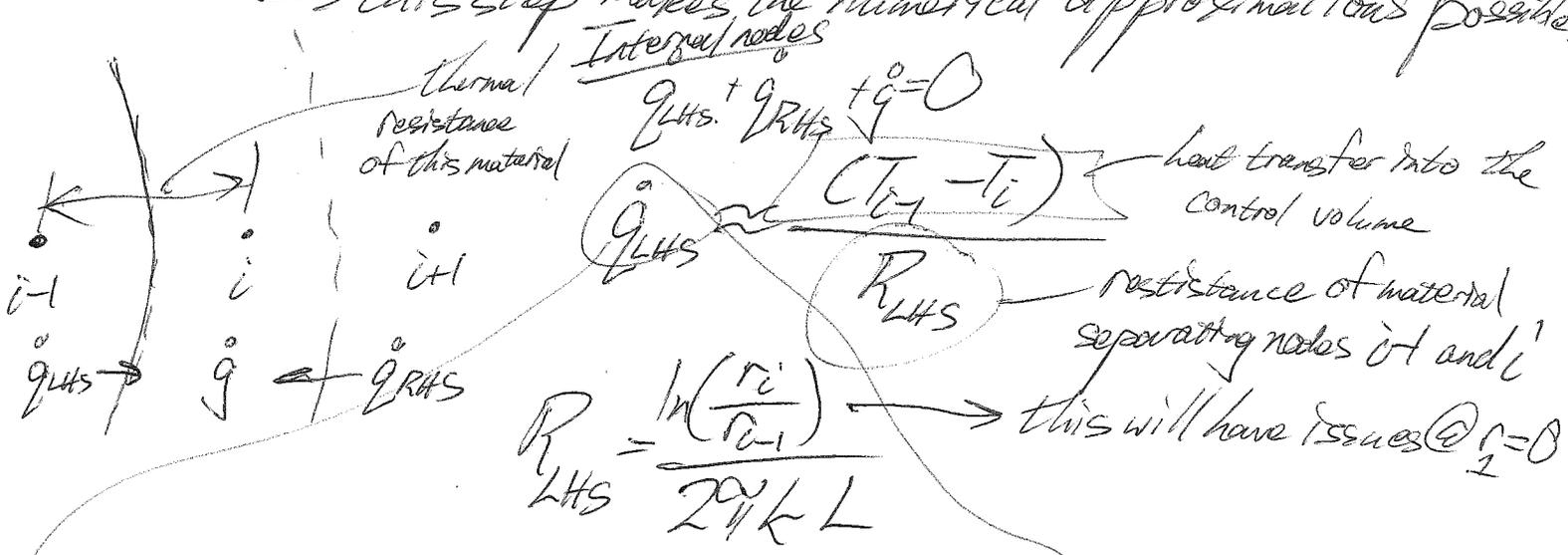


Internal nodes:  $IN = OUT + STORED$  → this is exactly correct energy must always balance

$$q_{LHS} + q_{RHS} + \dot{q} = 0$$

→ It doesn't matter whether heat transfers in or out of CV as long as I am consistent.

Step 3: Approximate each energy term with a rate equation  
 → this step makes the numerical approximations possible.



$$R_{LHS} = \frac{\ln\left(\frac{r_i}{r_{i-1}}\right)}{2\pi k L}$$

→ this will have issues @  $r_i = 0$

Alternative approximation:  $R_{LHS} = \frac{\text{conduction length}}{k(\text{conduction area})} = \frac{\Delta r}{k 2\pi L \left(r_i - \frac{\Delta r}{2}\right)}$

→ must be evaluated @ the interface

$$\dot{q}_{LHS} = \frac{2\pi L k \left(r_i - \frac{\Delta r}{2}\right) (T_{i-1} - T_i)}{\Delta r}$$

$$\dot{q}_{RHS} = \frac{2\pi L k \left(r_i + \frac{\Delta r}{2}\right) (T_{i+1} - T_i)}{\Delta r}$$

$$\dot{q}_{LHS} + \dot{q}_{RHS} + \dot{q} = 0$$

$$\dot{q} = \dot{q}''' \frac{2\pi r_i L \Delta r}{2}$$

→ For all internal nodes this is  $N-2$  equations in  $N$  unknowns  $(T_1, \dots, T_N)$

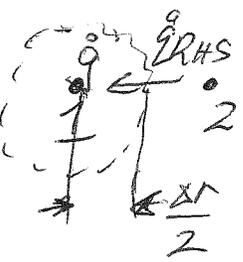
→ The remaining 2 eqs. must come from the boundary nodes

Boundary Node 1:

Energy Balance  $\dot{q}_{RHS} + \dot{q} = 0$  Rate Equations:

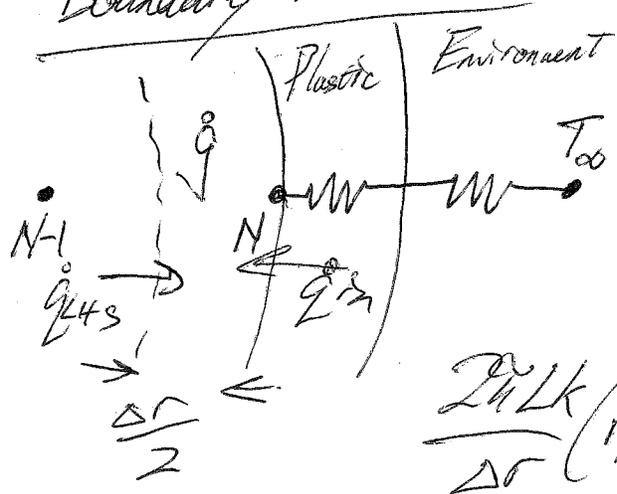
$$\dot{q}_{RHS} = \underbrace{2\pi L \left(\frac{\Delta r}{2}\right)}_{\text{Surface area}} \frac{k}{\Delta r} \underbrace{(T_2 - T_1)}_{\Delta T}$$

$$\dot{q} = \underbrace{\frac{2\pi L \left(\frac{\Delta r}{2}\right)^2}_{\text{Volume}}}_{\text{Volume}} \dot{q}'''$$



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Boundary Node N:



Energy Balance:  $\dot{q}_{LHS} + \dot{q}_m + \dot{q} = 0$

Rate Equations:  $\dot{q}_{LHS} = 2HL \left( N - \frac{\Delta r}{2} \right) \frac{k}{\Delta r} (T_{N-1} - T_N)$

$\dot{q}_m = \frac{(T_{\infty} - T_N)}{R_{cond,p} + R_{conv}}$

$\dot{q} = 2HL \left( N - \frac{\Delta r}{2} \right) q''_{conv}$

$\frac{2HLk}{\Delta r} \left( N - \frac{\Delta r}{2} \right) (T_{N-1} - T_N) + \frac{(T_{\infty} - T_N)}{R_{cond,p} + R_{conv}} + 2HL \left( N - \frac{\Delta r}{2} \right) q''_{conv} = 0$

Now we have a system of many algebraic equations (rather than a differential equation). It's easy to enter & solve these in EES,

Step 4: Implement model

Step 5: Verify that your model has numerically converged

- Do you have enough Nodes? → Examine an import solution metric as N changes (parametric table)
- Solution should stop changing as N becomes significantly large
- Decision requires engineering judgement,

Step 6: Verify your solution makes sense,

→ Show

Now for varying k & q, look @ where they are used.  $q$  is @ Node i, k is between nodes @ interface

$q''_0 = 0.001 T_i - 1$     $k_{LHS,i} = 0.00005 \left( \frac{T_i + T_0}{2} \right)^{1.3841}$     $k_{RHS,i} = 0.00005 \left( \frac{T_i + T_0}{2} \right)^{1.3841}$